

Spatially-discretised Distributed Port-Hamiltonian Systems Lumping, model reduction & control synthesis

Workshop on Model Reduction and Transport-dominated Phenomena

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IL PRESENTE MATERIALE È RISERVATO AL PERSONALE DELL'UNIVERSITÀ DI BOLOGNA E NON PUÒ ESSERE UTILIZZATO AI TERMINI DI LEGGE DA ALTRE PERSONE O PER FINI NON ISTITUZIONALI



This talk is organised into three main parts

- Structure-preserving Spatial Discretisation of Distributed Port-Hamiltonian Systems
- Basic Results on Model Reduction for Port-Hamiltonian Systems
- * Control Synthesis for Implicit Port-Hamiltonian Systems
- These topics are strongly related, and they show how it is possible to treat different problems within the same *methodological framework*
- Some academic examples are presented, and also one real-world industrial application





Structure-preserving Spatial Discretisation of Distributed Port-Hamiltonian Systems



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A fundamental problem in the simulation and control of systems containing distributed-parameter components concerns finitedimensional approximation

- *Numerical methods usually assume the boundary conditions to be given;
- Finite dimensional approximation methods are not easily relatable to numerical techniques for solving PDEs,
- * and are mainly confined to *linear PDEs*
- We propose a method for spatial discretisation of boundary control systems based on a particular type of *finite elements*
- The spatially discretised system is **again** a *port-Hamiltonian system*

Examples:

- Ideal transmission line;
- *Two dimensional wave equation;
- *Nonlinear flexible link



Spatial discretisation of distributed nH evetome G. Golo, et al., Automatica, 2004

WLet us start from a system of *two conservation laws*:

*Bond space:

$$Z \subset \mathbb{R}^{n} \quad f_{x} = \begin{pmatrix} f_{q} \\ f_{p} \end{pmatrix} \in \mathcal{F}_{x} = \Omega^{q}(Z) \times \Omega^{p}(Z) \quad x = \begin{pmatrix} q \\ p \end{pmatrix} \in \mathcal{X} = \Omega^{q}(Z) \times \Omega^{p}(Z)$$

$$e_{x} = \begin{pmatrix} e_{q} \\ e_{p} \end{pmatrix} \in \mathcal{E}_{x} = \Omega^{n-q}(Z) \times \Omega^{n-p}(Z)$$

$$\begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix} \in \mathcal{F}_{\partial} \times \mathcal{E}_{\partial} = \Omega^{n-p}(\partial Z) \times \Omega^{n-q}(\partial Z)$$

* Stokes-Dirac structure:

$$\begin{pmatrix} f_q \\ f_p \end{pmatrix} = \begin{pmatrix} 0 & -d \\ (-1)^{pq} d & 0 \end{pmatrix} \begin{pmatrix} e_q \\ e_p \end{pmatrix} \qquad \begin{pmatrix} f_{\partial} \\ e_{\partial} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ -(-1)^q & 0 \end{pmatrix} \begin{pmatrix} e_q |_{\partial Z} \\ e_p |_{\partial Z} \end{pmatrix}$$

with respect to the *scalar pairing*

$$\int_{Z} \left(e_q \wedge f_q + e_p \wedge f_p \right) + \int_{\partial Z} e_{\partial} \wedge f_{\partial}$$

$$f_{q} = \frac{\partial q}{\partial t} \quad f_{p} = \frac{\partial p}{\partial t}$$
$$e_{q} = \frac{\delta H}{\delta q} \quad e_{p} = \frac{\delta H}{\delta p}$$

******Energy balance*:

$$\frac{\mathrm{d}H}{\mathrm{d}t}(t) + \int_{\partial Z} e_{\partial} \wedge f_{\partial} = 0$$

 \mathbf{V} For a *transmission line*, $\mathbf{n} = \mathbf{q} = \mathbf{p} = \mathbf{1}$ and

$$p \equiv \phi \qquad H(q,\phi) = \frac{1}{2} \int_{Z} \left[\frac{\star q(z)}{C(z)} q(z) + \frac{\star \phi(z)}{L(z)} \phi(z) \right]$$

The spatial discretisation procedure consists of *two steps:*

* First, the *interconnection structure* is spatially discretised;

* Secondly, the *constitutive relations* are approximated

STEP#1: spatial discretisation of the interconnection structure

*Consider a part of the transmission line between two points a and b: the spatial manifold corresponding to this part of line is $Z_{ab} = [a, b]$

Assumption 1. Approximation
of
$$f_q$$
 and f_{ϕ} on Z_{ab}
 $f_q(t, z) = f_{q,ab}(t)\omega_{q,ab}(z)$
 $f_{\phi}(t, z) = f_{\phi,ab}(t)\omega_{\phi,ab}(z)$
where
 $\int_{Z_{ab}} \omega_{q,ab} = \int_{Z_{ab}} \omega_{\phi,ab} = 1$

Assumption 2. Approximation of e_q and e_{ϕ} on Z_{ab} $e_q(t, z) = e_{q,a}(t)\omega_{q,a}(z) + e_{q,b}(t)\omega_{q,b}(z)$ $e_{\phi}(t, z) = e_{\phi,a}(t)\omega_{\phi,a}(z) + e_{\phi,b}(t)\omega_{\phi,b}(z)$ where $\omega_{q,a}(a) = \omega_{q,b}(b) = \omega_{\phi,a}(a) = \omega_{\phi,b}(b) = 1$ $\omega_{q,a}(b) = \omega_{q,b}(a) = \omega_{\phi,a}(b) = \omega_{\phi,a}(b) = 0$

Z = [0, L]





 \mathbf{V} Integration over Z_{ab} leads to

$$f_{q,ab}(t) = e_{\phi,a}(t) - e_{\phi,b}(t)$$
$$f_{\phi,ab}(t) = e_{q,a}(t) - e_{q,b}(t)$$

$$\begin{pmatrix} e_{\partial,a} \\ e_{\partial,b} \\ f_{\partial,a} \\ f_{\partial,b} \\ f_{q,ab} \\ f_{\phi,ab} \end{pmatrix} = \begin{pmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & -1 \\ 1 & -1 & 0 & 0 \end{pmatrix} \begin{pmatrix} e_{q,a} \\ e_{q,b} \\ e_{\phi,a} \\ e_{\phi,b} \end{pmatrix}$$

Met power of the considered part of the transmission line:

$$P_{ab}^{\text{net}} = \int_{Z_{ab}} e_q(z)f_q(z) + \int_{Z_{ab}} e_{\phi}(z)f_{\phi}(z) - e_{\partial,a}f_{\partial,a} + e_{\partial,b}f_{\partial,b}$$

$$P_{ab}^{\text{net}} = \left[\alpha_{ab}e_{q,a} + (1 - \alpha_{ab})e_{q,b}\right]f_{q,ab} + \left[(1 - \alpha_{ab})e_{\phi,a} + \alpha_{ab}e_{\phi,b}\right]f_{\phi,ab} - \left[0 < \alpha_{ab} < 1\right]$$

$$e_{q,ab} = \alpha_{ab}e_{q,a} + (1 - \alpha_{ab})e_{q,b} + \left[(1 - \alpha_{ab})e_{\phi,a} + \alpha_{ab}e_{\phi,b}\right]f_{\phi,ab} - \left[0 < \alpha_{ab} < 1\right]$$

$$P_{ab}^{\text{net}} = e_{q,ab}f_{q,ab} + e_{\phi,ab}f_{\phi,ab} - e_{\partial,a}f_{\partial,a} + e_{\partial,b}f_{\partial,b}$$



STEP#2: approximation of the energy part

 e_q

*Since f_q and f_{ϕ} are related to q and ϕ , it is consistent to impose

$$q(t,z) = Q_{ab}(t)\omega_{q,ab}(z) \qquad \phi(t,z) = \Phi_{ab}(t)\omega_{\phi,ab}(z)$$

$$\int_{Z_{ab}} \frac{dQ_{ab}(t)}{2C(z)}q(t,z)$$
The approximation of the electric energy of the part of the line is
$$H_{q,ab}(Q_{ab}(t)) = \frac{1}{2}\frac{Q_{ab}^2}{C_{ab}} \qquad C_{ab}^{-1} = \int_{Z_{ab}} \star \left(\frac{\omega_{q,ab}(z)}{C(z)}\right)\omega_{q,ab}(z)$$
Similarly, the magnetic energy is approximated by:
$$H_{\phi,ab}(\Phi_{ab}(t)) = \frac{1}{2}\frac{\Phi_{ab}^2}{L_{ab}} \qquad L_{ab}^{-1} = \int_{Z_{ab}} \star \left(\frac{\omega_{\phi,ab}(z)}{L(z)}\right)\omega_{\phi,ab}(z)$$
Total energy:
$$H_{ab}(Q_{ab}, \Phi_{ab}) = H_{q,ab}(Q_{ab}) + H_{\phi,ab}(\Phi_{ab})$$

$$e_{\phi,ab}(t) = \frac{\partial H_{ab}(Q_{ab}, \Phi_{ab})}{\partial \Phi_{ab}}(t) = \frac{\Phi_{ab}(t)}{L_{ab}}$$

Spatial discretisation of the transmission line:

$$Q = \begin{pmatrix} Q_{S_0S_1} & Q_{S_1S_2} & \cdots & Q_{S_{n-1}S_n} \end{pmatrix}^{\mathrm{T}} \\ \Phi = \begin{pmatrix} \Phi_{S_0S_1} & \Phi_{S_1S_2} & \cdots & \Phi_{S_{n-1}S_n} \end{pmatrix}^{\mathrm{T}}$$

- *The transmission line is split into *n parts*
- * The *i*-th part (S_{i-1} ; S_i) is discretised as explained in the previous two subsections, where $a = S_{i-1}$ and $b = S_i$
- Since the interconnection of port-Hamiltonian systems is a port-Hamiltonian system, the total discretised system is also a port-Hamiltonian system
- * The *total Hamiltonian* is given by the sum of the individual Hamiltonians

$$H(Q,\Phi) = \sum_{i=1}^{n} \frac{Q_{S_{i-1}S_i}^2}{2C_{S_{i-1}S_i}} + \sum_{i=1}^{n} \frac{\Phi_{S_{i-1}S_i}^2}{2L_{S_{i-1}S_i}}$$

******Energy balance*:

$$\frac{\mathrm{d}H(Q(t),\Phi(t))}{\mathrm{d}t} - e_{\partial,0}f_{\partial,0} + e_{\partial,L}f_{\partial,L} = 0$$

boundary port

$$(f_{\partial,S_0}, e_{\partial,S_0}) = (f_{\partial,0}, e_{\partial,0})$$

 $(f_{\partial,S_n}, e_{\partial,S_n}) = (f_{\partial,L}, e_{\partial,L})$



Two-dimensional case: the wave equation $\mu \ddot{u} + E\Delta u = 0, \qquad u(t, z) \in \mathbb{R}, \ z = (z_1, z_2) \in Z$

Port-Hamiltonian description:

*State (energy) variables:

$$p(t, z_1, z_2) \in \Omega^2(Z)$$
 $\epsilon(t, z_1, z_2) = \frac{\partial u}{\partial z_1} dz_1 + \frac{\partial u}{\partial z_2} dz_2 \in \Omega^1(Z)$

***** Hamiltonian function:

$$H(p,\epsilon) = \int_{Z} \mathcal{H}(p,\epsilon), \qquad \mathcal{H}(p,\epsilon) = \frac{1}{2} \left(p \wedge v + \epsilon \wedge \sigma \right)$$

***** *Port-Hamiltonian formulation:*

$$\begin{pmatrix} \dot{\epsilon} \\ \dot{p} \end{pmatrix} = \begin{pmatrix} 0 & d \\ -d & 0 \end{pmatrix} \begin{pmatrix} \frac{\delta H}{\delta \epsilon} \\ \frac{\delta H}{\delta p} \end{pmatrix} \qquad \begin{pmatrix} v_{\partial} \\ \sigma_{\partial} \end{pmatrix} = \begin{pmatrix} 0 & 1 \\ 1 & 0 \end{pmatrix} \begin{pmatrix} \frac{\delta H}{\delta \epsilon} |_{\partial Z} \\ \frac{\delta H}{\delta p} |_{\partial Z} \end{pmatrix}$$

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 $\sigma = E \star \epsilon$

Passing from the interval grid for the 1-D case, we move onto the simplest possible grid for the 2-D example, i.e. the triangular grid

* Approximation of *flow variables*:

 $f_p(t,z) = f_{p,abc}(t)\omega_{p,abc}(z)$

 $f_{\epsilon}(t,z) = f_{\epsilon,ab}(t)\omega_{\epsilon,ab}(z) + f_{\epsilon,bc}(t)\omega_{\epsilon,bc}(z) + f_{\epsilon,ca}(t)\omega_{\epsilon,ca}(z)$ ***** Approximation of *efforts*:

 $e_{v}(t,z) = e_{v,a}(t)\omega_{v,a}(z) + e_{v,b}(t)\omega_{v,b}(z) + e_{v,c}(t)\omega_{v,c}(z)$ $e_{v}(t,z) = e_{v,a}(t)\omega_{v,a}(z) + e_{v,b}(t)\omega_{v,b}(z) + e_{v,c}(t)\omega_{v,c}(z)$

 $e_{\sigma}(t,z) = e_{\sigma,ab}(t)\omega_{\sigma,ab}(z) + e_{\sigma,bc}(t)\omega_{\sigma,bc}(z) + e_{\sigma,ca}(t)\omega_{\sigma,ca}(z)$ ***** Boundary variables:

$$f_{\partial}(t,z) = f_{\partial,a}(t)\omega_{v,a}(z) + f_{\partial,b}(t)\omega_{v,b}(z) + f_{\partial,c}(t)\omega_{v,c}(z)$$
$$e_{\partial}(t,z) = e_{\partial,ab}(t)\omega_{\sigma,ab}(z) + e_{\partial,bc}(t)\omega_{\sigma,bc}(z) + e_{\partial,ca}(t)\omega_{\sigma,ca}(z)$$

$$\int_{Z_{abc}} \omega_{p,abc} = 1$$

$$\int_{U} \omega_{\epsilon/\sigma,l} = \begin{cases} 0 & \text{if } l \neq l' \\ 1 & \text{if } l = l' \\ l, l' \in \{ab, bc, ca\} \end{cases}$$

$$\omega_{v,x}(y) = \begin{cases} 0 & \text{if } x \neq y \\ 1 & \text{if } x = y \\ x, y \in \{a, b, c\} \end{cases}$$

$$ab Z_{abc}$$

$$ab Z_{abc}$$

b,

Similarly to the 1D case and after some time, you obtain the following relation, that defines a *Dirac structure*:



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Spatial discretisation of di $p(t,z) = p_{abc}(t)\omega_{p,abc}(z)$ $\epsilon(t,z) = \rho_{abc}(t)\omega_{p,abc}(z)$ $\epsilon(t,z) = \epsilon_{ab}(t)\omega_{\epsilon,ab}(z) + \epsilon_{bc}(t)\omega_{\epsilon,bc}(z) + \epsilon_{ca}(t)\omega_{\epsilon,ca}(z)$

Approximation of the constitutive relations:

**Kinetic energy:*

$$H_{p,abc}(p) = \frac{1}{2} \int_{Z_{abc}} \star \frac{p(t,z)}{\mu(z_1,z_2)} \wedge p(t,z)$$
$$= \frac{p_{abc}^2(t)}{2} \underbrace{\int_{Z_{abc}} \frac{\star \omega_{p,abc}(z) \wedge \omega_{p,abc}(z)}{\mu(z)}}_{M^{-1}} = \frac{p_{abc}^2(t)}{2M}$$

* Potential elastic energy:

$$H_{\epsilon,abc}(\epsilon) = \frac{1}{2} \int_{Z_{abc}} \frac{\star \epsilon}{Y(z)} \wedge \epsilon = \frac{1}{2} \left[\frac{\epsilon_{ab}^2}{Y_1} + \frac{\epsilon_{ca}^2}{Y_2} - \frac{2\epsilon_{ab}\epsilon_{ca}}{Y_3} \right]$$

$$\epsilon(t,z) = \epsilon_{ab}(t) d\omega_{v,b}(z) - \epsilon_{ca} d\omega_{v,c}(z) + [\epsilon_{ab}(t) + \epsilon_{bc}(t) + \epsilon_{ca}(t)] \omega_{\epsilon,bc}$$

$$Y_1^{-1} = \int_{Z_{abc}} \frac{\star d\omega_{v,b}(z) \wedge d\omega_{v,b}(z)}{Y}$$

$$Y_2^{-1} = \int_{Z_{abc}} \frac{\star d\omega_{v,c}(z) \wedge d\omega_{v,c}(z)}{Y}$$

$$Y_3^{-1} = \int_{Z_{abc}} \frac{\star d\omega_{v,b}(z) \wedge d\omega_{v,c}(z) + \star d\omega_{v,c}(z) \wedge d\omega_{v,b}(z)}{2Y}$$



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The algebraic coupling is easily tackled by defining $\tilde{f}_q = f_q + \operatorname{ad}_{q+\hat{n}} e_p$ $\tilde{f}_p = f_p - \operatorname{ad}_{q+\hat{n}}^* e_q$

In this way, the Stokes-Dirac structure of the transmission line appears and the boundary conditions are not changed



The constitutive equations are treated in the usual way

* *Kinetic energy*:



$$\mathcal{U}(q_{ab}) = \frac{1}{2} \langle q_{ab} \mid q_{ab} \rangle_{C^{-1}} \int_{Z_{ab}} \star \omega_{q,ab}(z) \omega_{q,ab}(z)$$
$$= \frac{1}{2} \langle q_{ab} \mid q_{ab} \rangle_{C^{-1}_{ab}}$$

the complete model follows again via power conserving interconnection of such "simpler" elements



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Mow the discretisation procedure works? Is it accurate and does it assure a certain convergence as far as the number N of finite elements increases?

* The model is non-linear, so it is not possible to perform a spectral analysis nor compute an exact solution

*...but for "small deformation" it is equivalent to the *Timoshenko beam*

Μ	Eigenfrequencies $\omega_1, \ldots, \omega_6$ [rad/s]					
6	8.4085	50.8229	141.7407	293.6579	532.5828	579.5340
9	8.3693	48.2608	124.5600	228.5422	365.4776	529.2416
12	8.3556	47.4113	119.3055	210.9628	321.2388	449.7060
15	8.3492	47.0260	116.9909	203.5459	303.5649	415.1332
18	8.3457	46.8187	115.7633	199.6910	294.6171	397.8813
21	8.3436	46.6945	115.0329	197.4233	289.4292	387.9998
24	8.3423	46.6141	114.5628	195.9736	286.1422	381.7907
∞ (exact)	8.3378	46.3530	113.0483	191.3562	275.8244	362.5837

eigenfrequencies in the free-clamped case

Simulation: flexible link



Simulation: complete system



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Basic Results on Model Reduction for Port-Hamiltonian Systems



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The idea is to illustrate a novel procedure for the *model reduction of high-order port-Hamiltonian systems*

- The method can be applied to port-Hamiltonian systems not necessarily in input-state-output form
- A typical application is an high-order systems obtained from the spatial discretisation of distributed port-Hamiltonian systems
 - *The approximating system turns out to be completely *a-causal* and able to deal with *time varying boundary conditions*
 - * "A-causality" implies that (boundary) inputs and outputs are determined by the causality of the interconnected subsystems
 - * The plant dynamics are given as a set of DAEs
- The model-reduction technique is able to deal with such systems without loosing the port-Hamiltonian structure and related properties



- *****With *constant* Dirac structure;
- ******Quadratic* Hamiltonian function

$$\mathcal{D} = \left\{ (f_S, f_C, f_I, e_S, e_C, e_I) \in \mathcal{F} \times \mathcal{E} \mid \\ \mid F_S f_S + F_C f_C + F_I f_I + \\ + E_S e_S + E_C e_C + E_I e_I = 0 \right\}$$



$$H(x) = \frac{1}{2}x^{\mathrm{T}}Lx$$

Solution State State

$$A_f^{\mathrm{T}} f_S = 0 \qquad \qquad A_e^{\mathrm{T}} e_S = 0$$

Т

The reduced order model is obtained once the constraints have been explicitly removed from the system dynamics

$$S = \operatorname{Ker} A$$

$$S^{T}S = I_{r}$$

$$A_{e}^{T}A_{e} =$$

$$A_{f}^{T}A_{f} =$$

$$A_{e}^{T}A_{f} =$$



The reduction step is performed by using S as projection matrix on the Dirac structure and Hamiltonian function of the full-order system

$$\tilde{f}_{S} = \begin{pmatrix} f_{S,e} \\ \tilde{f}_{S,f} \\ \tilde{f}_{S,r} \end{pmatrix} = \begin{pmatrix} A_{e}^{\mathrm{T}} \\ A_{f}^{\mathrm{T}} \\ S^{\mathrm{T}} \end{pmatrix} f_{S}$$

$$coordinate change$$

$$\tilde{e}_{S} = \begin{pmatrix} \tilde{e}_{S,e} \\ \tilde{e}_{S,f} \\ \tilde{e}_{S,r} \end{pmatrix} = \begin{pmatrix} A_{e}^{\mathrm{T}} \\ A_{f}^{\mathrm{T}} \\ S^{\mathrm{T}} \end{pmatrix} e_{S}$$

$$T = \begin{pmatrix} A_{e} & A_{f} & S \end{pmatrix}$$

In these new coordinates the constraints can be expressed as

$$\tilde{f}_{S,f} = 0 \qquad \qquad \tilde{e}_{S,e} = 0$$

MThese conditions *fix the causality* on the corresponding power ports

*Since these port variables do not play any role in the energy balance, *they can be removed* to obtain the reduced order Dirac structure



Simple computations show that the *reduced Dirac structure* is $F_{S,r}\tilde{f}_{S,r} + F_{C,r}f_C + F_{I,r}f_I + E_{S,r}\tilde{e}_{S,r} + E_{C,r}e_C + E_{I,r}e_I = 0$

with

$$F_{S,r} = G^{\perp}F_SS \quad F_{C,r} = G^{\perp}F_C \quad F_{I,r} = G^{\perp}F_I$$
$$E_{S,r} = G^{\perp}E_SS \quad E_{C,r} = G^{\perp}E_C \quad E_{I,r} = G^{\perp}E_I$$



The coordinate change on the storage flows induces a similar transformation on the energy variables and on the Hamiltonian:

$$\tilde{x} = \begin{pmatrix} \tilde{x}_e \\ \tilde{x}_f \\ \tilde{x}_r \end{pmatrix} = \begin{pmatrix} A_e^{\mathrm{T}} \\ A_f^{\mathrm{T}} \\ S^{\mathrm{T}} \end{pmatrix} x = T^{-1}x \qquad \tilde{H}(\tilde{x}) = H(\tilde{x}) = \frac{1}{2}\tilde{x}^{\mathrm{T}}\tilde{L}\tilde{x}$$
$$\tilde{x}_f = \kappa \qquad \qquad \tilde{d}\tilde{H} = 0$$

The condition on the gradient of the Hamiltonian implies that



The constraint on the flow is easily handled and leads to

$$\tilde{H}_{r}(\tilde{x}_{r}) = \frac{1}{2} \tilde{x}_{r}^{\mathrm{T}} \tilde{L}_{rr} \tilde{x}_{r}$$

$$\tilde{f}_{S,r} = -\dot{\tilde{x}}_{r} \qquad \tilde{e}_{S,r} = \frac{\partial \tilde{H}_{r}}{\partial \tilde{x}_{r}} = \tilde{L}_{rr} \tilde{x}_{r}$$

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Application to a piezo-active structure: the activity aimed at developing a model reduction procedure able to preserve the frequency behaviour of the original system in a neighbourhood of a predefined set of frequencies of interest

- *This research is carried out in collaboration with TetraPak and deals with modelling and simulation of the *Ultrasonic Sealing System (USS)*
- ☑Due to the presence of a Compact Transducer (CT), it is not possible to perform a simulation of the complete system to
 - * Test the validity of the controller;
 - * Perform the diagnosis of the sealing process in detail
- The proposed procedure drastically reduces the simulation time without loosing the essential dynamical information



- The Ultrasonic Sealing System (USS) is a key technology used in the sealing process and based on ultrasounds for multi-layered packaging materials
 - *The most complex part of the USS is the a Compact Transducer (CT) which is responsible of the sealing process
 - * Physically, it is a piezo-electric actuator excited with high frequency sinusoidal inputs





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The f.e.m. of a mechanical system embedding piezo-electric actuators and sensors is

$$\begin{pmatrix} M_{ww} & 0 \\ 0 & 0 \end{pmatrix} \begin{pmatrix} \ddot{w} \\ \ddot{\phi} \end{pmatrix} + \begin{pmatrix} K_{ww} & K_{\phi w}^{\mathrm{T}} \\ K_{\phi w} & K_{\phi \phi} \end{pmatrix} \begin{pmatrix} w \\ \phi \end{pmatrix} = \begin{pmatrix} F \\ G \end{pmatrix}$$

The *inputs* of the system are *forces F* and *voltages* ϕ_C , which represents the effective input voltage over the electric d.o.f.

$$\phi = B_C \phi_C + B_F \phi_F$$

Since there is no electric charge imposed on the free electrical d.o.f., it is possible to "eliminate" ϕ_F



Oue to the particular operative conditions of this device, the reduced order model preserves the behaviour of the original system in a neighbourhood of a predefined set of frequencies of interest

*Solve an *eigenvalue/eigenvector problem* for *k* eigenvalues in a neighbourhood of the specified set

$$\bar{S} = (\bar{S}_1 \quad \cdots \quad \bar{S}_k) \quad \triangleright \quad \bar{A} = \operatorname{Ker} \bar{S}^{\mathrm{T}} \quad \triangleright \quad (\bar{A}^{\mathrm{T}} \dot{q} = 0)$$

The result is a set of independent constraints on both the flows and the efforts of the full-order dynamics

$$A_f = \begin{pmatrix} \bar{A} \\ 0 \end{pmatrix} \qquad \qquad A_e = \begin{pmatrix} 0 \\ \bar{A} \end{pmatrix}$$

Simple computations show that

$$\tilde{L}_{rr} = \begin{pmatrix} K_r & 0\\ 0 & M_r^{-1} \end{pmatrix} = \begin{pmatrix} \bar{S}^{\mathrm{T}} K \bar{S} & 0\\ 0 & \left(\bar{S}^{\mathrm{T}} M \bar{S} \right)^{-1} \end{pmatrix}$$

⊠Abaqus FEM CAD software has been used

☑A sufficiently fine mesh have been designed:

*****128322 nodes with **389286 d.o.f.**

Reduction platform:

*Core i7 940 (4 core + 4ht core) 2.93 Mhz

12 Gb DDR3 Ram

- Linux ubuntu 9.04 64bit
- *Matlab 2009a

Reduction algorithm runs in 17 min

☑ Abaqus FEM CAD is able to simulate 0.5 ms in 24 hours

Max step size 10⁻⁷ sec;

*****Complete sealing phase 150 ms; **1 year to perform a simulation!**

Reduced order model considering 10 eigenvalues around 28 KHz is able to simulate 150 ms in 30 sec



Simulation results

- *Sinusoidal input: 100 V @28640 Hz, 0÷10 ms
- Head displacement



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S Experimental results



Measured and simulated displacement during the sealing process of the center of the head

Measured and simulated displacement during the sealing process of the top of the screw the piezo-electric actuator

Measured and simulated input voltage

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Control Synthesis for Implicit Port-Hamiltonian Systems



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 \mathcal{K} (dissipation)

Dirac structures

☑The idea is to develop a general theory for the energy-based control of implicit port-Hamiltonian systems, i.e. written as a set of DAEs

☑ Denote by 𝔅 × 𝔅 the space of *power variables*, and by $\langle e, f \rangle$ the *power* associated to the port (*f*, *e*) ∈ 𝔅 × 𝔅

Operation of a structure of a linear subspace $\mathcal{O} \subset \mathcal{P} \times \mathcal{E}$ such that

$$\dim \mathcal{D} = \dim \mathcal{F} \qquad \qquad \langle e, f \rangle = 0, \ \forall (f, e) \in \mathcal{D}$$

☑Coordinate representations:

$$\begin{array}{l} \overbrace{\text{finge}} \mathcal{D} = \left\{ (f,e) \in \mathcal{F} \times \mathcal{E} \mid f = E^{\mathrm{T}}\lambda, \ e = F^{\mathrm{T}}\lambda, \ \lambda \in \mathbb{R}^{n} \right\} \quad \begin{array}{l} EF^{\mathrm{T}} + FE^{\mathrm{T}} = 0 \\ \operatorname{rank} (F \mid E) = n \end{array} \\ \overbrace{\text{finder}} \mathcal{D} = \left\{ (f,e) \in \mathcal{F} \times \mathcal{E} \mid Ff + Ee = 0 \right\} \\ \overbrace{\text{finder}} f = -\dot{x} \ e = \frac{\partial H}{\partial x} \\ -F\dot{x} + E \frac{\partial H}{\partial x} = 0, \quad x(0) = x_{0} \in \mathcal{X} \end{aligned}$$

$$\begin{array}{l} D = \left\{ (f,e) \in \mathcal{F} \times \mathcal{E} \mid Ff + Ee = 0 \right\} \\ \overbrace{\text{finder}} f = -\dot{x} \ e = \frac{\partial H}{\partial x} \\ -F\dot{x} + E \frac{\partial H}{\partial x} = 0, \quad x(0) = x_{0} \in \mathcal{X} \end{aligned}$$

Dirac structures & port-Hamiltonian systems



Dirac structures & port-Hamiltonian systems

Example: the RLC circuit



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Energy-based control

Standard approach is to rely on *"energy considerations"* to obtain and prove asymptotic stability of equilibria

* Damping injection

* Energy-shaping

Standard assumption is *H* bounded from below



Damping injection

Suppose that *H* has an isolated minimum at a desired equilibrium

$$\frac{\partial H}{\partial x}(x^{\star}) = 0 \qquad \qquad \frac{\partial^2 H}{\partial x^2}(x^{\star}) > 0$$

The idea is to *dissipate energy* until the minimum is reached

- *Asymptotic stability if there is "enough dissipation"
- Zero-state detectability
- *La Salle's Invariance principle

The control action is

$$u(t) = -K_D y(t), \qquad K_D = K_D^{\mathrm{T}} \ge 0$$

$$\frac{\mathrm{d}H}{\mathrm{d}t}(x(t)) = -\left(\frac{\partial H}{\partial x}(x(t))\right)^{\mathrm{T}} \left(R(x(t)) + K_D\right) \frac{\partial H}{\partial x}(x(t)) \le 0$$



☑In general, it is necessary to shape the open-loop Hamiltonian to introduce a minimum at the desired equilibrium

From the energy-balance relation we have

$$H(x(t)) - H(x(0)) = \int_0^t y^{\mathrm{T}}(\tau) u(\tau) \,\mathrm{d}\tau - d(t)$$

The standard formulation of passivity-based control requires to determine a control action

$$u(t) = \beta(x(t)) + u'(t)$$

such that the *closed-loop dynamics* satisfies:

$$H_d(x(t)) - H_d(x(0)) = \int_0^t {y'}^{\mathrm{T}}(\tau) u'(\tau) \,\mathrm{d}\tau - d_d(t)$$



Image: Contract Advised Adv

* Energy-shaping *plus* damping injection



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Energy-balancing control

A large class of dynamical systems can be stabilised by requiring that the supplied energy is a function of the state of the plant

$$-\int_0^t y^{\mathrm{T}}(\tau)\beta(x(\tau))\,\mathrm{d}\tau = H_a(x(t)) + \kappa$$

We require that along *all* system trajectories

$$-y^{\mathrm{T}}(t)\beta(x(t)) = \frac{\partial^{\mathrm{T}}H_a}{\partial x}(x(t))\dot{x}(t)$$

The *"desired" closed-loop Hamiltonian* is then $H_d(x(t)) = H(x(t)) + H_a(x(t))$



The previous PDE provides the class of H_a and the control actions, while stability analysis follows from the energy-balance relation

*u' can be used to add damping



The methodology can be applied to generic nonlinear systems

$$\begin{cases} \dot{x} = f(x) + g(x)u\\ y = h(x) \end{cases}$$

Solution From KYP lemma, passivity is equivalent to the existence of a function H(x) such that

$$\left(\frac{\partial H}{\partial x}(x)\right)^{\mathrm{T}} f(x) \le 0$$
 $h(x) = g^{\mathrm{T}}(x)\frac{\partial H}{\partial x}(x)$

Matching equation:

$$\left(\frac{\partial H_a}{\partial x}(x)\right)^{\mathrm{T}} \left[f(x) + g(x)\beta(x)\right] = -h^{\mathrm{T}}(x)\beta(x)$$

☑At the equilibrium:

$$f(x^{\star}) + g(x^{\star})\beta(x^{\star}) = 0 \quad \Rightarrow \quad h^{\mathrm{T}}(x^{\star})\beta(x^{\star}) = 0$$

dissipati



The idea is to compute a state feed-back action $u(t) = \beta(x(t)) + u'(t)$

so that *the open-loop system is mapped into a new one,* but with a desired Hamiltonian

$$H_d(x(t)) = H(x(t)) + H_a(x(t))$$
$$\dot{x}(t) = \left[J(x(t)) - R(x(t))\right] \frac{\partial H_d}{\partial x}(x(t)) + G(x(t))u'(t)$$

MA direct computation leads to

$$G(x)\beta(x) = \left[J(x) - R(x)\right]\frac{\partial H_a}{\partial x}(x) \quad \text{matching constraints}$$

A further generalisation leads to the IDA-PBC control technique, where we shape

*Hamiltonian

Interconnection and resistive structure

ndition



Dirac structures & control synthesis

Section Control in the general case:

$$\beta^{\mathrm{T}}(x(t))f_{C}(t) = \left(\frac{\partial H_{a}}{\partial x}(x(t))\right)^{\mathrm{T}}\dot{x}(t)$$
$$\left(-\frac{\partial^{\mathrm{T}} H_{a}}{\partial x}E_{S}^{\mathrm{T}} + \beta^{\mathrm{T}}E_{C}^{\mathrm{T}}\right)\lambda = 0$$
$$-E_{S}\frac{\partial H_{a}}{\partial x} + E_{C}\beta = 0$$

$$(y,u) = (f_C, e_C)$$

M (necessary and) sufficient condition is that

$$\begin{pmatrix} 0 \\ 0 \\ -\frac{\partial H_a}{\partial x} \\ 0 \\ \beta \end{pmatrix} \in \operatorname{Im} \begin{pmatrix} E_S^{\mathrm{T}} \\ E_R^{\mathrm{T}} \\ E_C^{\mathrm{T}} \\ F_S^{\mathrm{T}} \\ F_R^{\mathrm{T}} \\ F_R^{\mathrm{T}} \\ F_C^{\mathrm{T}} \end{pmatrix} \equiv \mathcal{D}$$
 dissipation obstacle?



Dirac structures & control synthesis

In the second consider at first the finite element model of a transmission line, which is characterised by a Dirac structure with matrices

 $F_{\infty} = \begin{pmatrix} F_{S,\infty} & F_{C,\infty} & F_{I,\infty} \end{pmatrix} \qquad E_{\infty} = \begin{pmatrix} E_{S,\infty} & E_{C,\infty} & E_{I,\infty} \end{pmatrix}$ and an Hamiltonian

$$H_{\infty}(x_{\infty}) = \frac{1}{2} \sum_{i=1}^{N} \left(\frac{x_{q}^{i^{2}}}{C_{i}} + \frac{x_{\phi}^{i^{2}}}{L_{i}} \right)$$
$$x_{\infty} = \left(x_{q}^{1} \quad x_{\phi}^{1} \quad \dots \quad x_{q}^{N} \quad x_{\phi}^{N} \right)^{\mathrm{T}}$$





The plant is a finite dimensional port-Hamiltonian system with control port (f_C , e_C)

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W For the complete system we have

$$x = \begin{pmatrix} x_Q & x_\Phi & x_\infty \end{pmatrix}^{\mathrm{T}} \stackrel{\text{def}}{=} H(x) = H_{\infty}(x_\infty) + H_L(x_Q, x_\Phi)$$

Simple physical considerations lead to the *desired equilibrium:*

$$\frac{x_Q^{\star}}{C_L} = \frac{x_q^{i,\star}}{C_i} = e^{\star} \qquad \qquad \frac{x_{\Phi}^{\star}}{L_L} = \frac{x_{\phi}^{i,\star}}{L_i} = 0$$

 \mathbf{V} The energy-balance controller follows if it exists λ such that

$$-\frac{\partial H_{a}}{\partial x} = F_{S}^{T}\lambda$$

$$\beta = F_{C}^{T}\lambda$$

$$0 = E_{S}^{T}\lambda = E_{R}^{T}\lambda = F_{R}^{T}\lambda = E_{C}^{T}\lambda$$

$$H_{a}(x) = H_{a}(\xi)$$

$$H_{a}(x) = \frac{1}{2}\frac{\xi^{2}}{C_{C}} - e^{\star}\left(1 + \frac{C_{L}}{C_{C}} + \sum_{i=1}^{N}\frac{C_{i}}{C_{C}}\right)\xi + \kappa$$

$$\beta(x) = -\frac{\partial H_{a}}{\partial\xi}$$

$$\xi = x_{Q} + \sum_{i=1}^{N}x_{q}^{i}$$



Finding the *EB regulator* means finding a state dependent control action able to shape the open-loop Hamiltonian, in such a way that closed loop and target dynamics have the *same behaviour at the storage, resistive and control ports*

- ***** Very strong requirement!
- *Let us ask less: *just a matching* between open-loop *plus* controller, and target dynamics (with desired stability properties)





Since trajectories are required to be the same

$$0 = E_{S}^{\mathrm{T}} (\lambda - \lambda')$$

$$-\frac{\partial H_{a}}{\partial x} = F_{S}^{\mathrm{T}} (\lambda - \lambda')$$

$$0 = \left(R_{f} E_{R}^{\mathrm{T}} + R_{e} F_{R}^{\mathrm{T}}\right) (\lambda - \lambda')$$

$$\beta = F_{C}^{\mathrm{T}} (\lambda - \lambda')$$

$$\int \left(\begin{array}{c} 0 \\ -\frac{\partial H_{a}}{\partial x} \\ 0 \\ \beta \end{array} \right) \in \mathrm{Im} \left(\begin{array}{c} E_{S}^{\mathrm{T}} \\ F_{S}^{\mathrm{T}} \\ R_{f} E_{R}^{\mathrm{T}} + R_{e} F_{R}^{\mathrm{T}} \\ F_{C}^{\mathrm{T}} \end{array} \right)$$

☑It is possible to prove that the open-loop system is mapped into the desired closed-loop one, for which the Hamiltonian function H_d is selected so that "nice" stability properties are satisfied

*Asymptotic stability follows as in case of energy-balancing regulators





Dirac structures & control synthesis



 \blacksquare A possible *choice for* H_a can be the following:

$$H_a(\xi) = \frac{1}{2} \frac{(\xi - \xi^*)^2}{L_C} - \frac{e^*}{R_L} \xi + \kappa$$

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Asymptotic stability is a consequence of the *energy dissipation inequality*

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DE'16, Bertinoro, Italy – June 13-15, 2016

2nd IFAC Workshop on Control of Systems Governed by Partial Differential Equations

INVITATION

The second IFAC Workshop on Control of Systems Governed by Partial Differential Equations will be held in Bertinoro (Italy) from Monday to Wednesday 13-15 June 2016 at the Centro Residenziale Universitario of the University of Bologna.

The workshop will address new and state-of-the-art developments in modelling and control of distributed parameter systems and their applications. Since the control design for these systems resides at the intersection of mathematics, systems and control theory, control systems technology, computer and information science, it is essential to provide a joint forum to foster and evolve this important and emerging field of research.

CPDE'16 aims at providing this forum under the IFAC flagship.

SCOPE

http://www.cpde2016.org

Distributed parameter systems, which are mathematically described by partial differential equations, impose a formidable challenge in many applications coming from classical industrial fields as well as emerging sectors related to energy, transport, communi-

cation or medical science. Herein, the distriessential ingredient of the modelling and ana tribution of the system variables cannot be r distributed parameter systems essentially reli

and estimation strategies to influence the system dynamics, and to enlarge the dynamic operating range.

Starting from these observations, new approaches to the control of distributed para-

Hosting institution University of Bologna Location

Centro Residenziale Universitario, on the hilltop of Bertinoro (FC), Italy

IFAC-



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IFAC International Federation of Automatic Control, IFAC TC Distributed Parameter Systems

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